## A RADIATOR SYSTEM FOR COOLING A SHORT

## CYLINDRICAL BODY

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A solution is given for the heat loss and optimal design of radial cones (solid and hollow) that lie in the same plane.

We have given [1] a solution for the design and optimization of a system of radial cones diverging from an isothermal sphere.

These relationships [1] represent a particular case of a short cylinder when the cones diverge in a plane instead of three dimensions (Fig. 1). In this case the length of the cylinder equals the base diameter of a cone, while the radius is determined by the condition of contact between the bases of the cones.

Figure 2 shows computer results from the equation of [1] for the system of Fig. 1 when the surfaces are black. The performance $\theta$ of the system in Fig. 2 is the ratio of the actual radiated flux to the limiting flux $Q_{l i m}$ that would be emitted if the cones had infinite thermal conductivity and if there were no radiative exchange between the cones:

$$
Q_{\mathrm{Iim}}=n \pi \operatorname{tg} \frac{\alpha}{2} \sigma T_{0}^{4} L^{2}
$$

These results are correct also for hollow cones lacking heat transfer between the internal surfaces and having a wall thickness $\delta$ that varies as follows along the axis:

$$
\frac{(L-x) \operatorname{tg} \frac{\alpha}{2}-\delta}{(L-x) \operatorname{tg} \frac{\alpha}{2}}=\varphi=\text { const. }
$$

In this case $N$ must be replaced by $N /\left(1-\varphi^{2}\right)$ in order to use the relationships of Fig. 2.


Fig. 1. Radiation system with cooling cones.

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Fig. 2. Efficiency of radiation system versus $\alpha$ and N for different numbers of cones with $\varepsilon=1$ (figures near curves are values of $n$ ).

The curves of Fig. 2 allow one not only to determine the heat flux from the sizes of the cones (together with $T_{0}$ and $\lambda$ ) but also to optimize the system of Fig. 1, which with hollow cones can sometimes be considerably better than the normal annular fin [2].

## NOTATION

is the emissivity of surface;
$\sigma$ is the Stefan-Boltzmann radiation constant;
$\lambda \quad$ is the thermal conductivity of material;
$L$ is the length of cone;
$\alpha \quad$ is the angle at the cone vertex;
n is the number of cones;
$\mathrm{T}_{0}$ is the temperature of cooled body surface;
$\delta \quad$ is the thickness of hollow cone wall at distance x from its base;
$\theta$ is the efficiency of heat-removing system;
$\mathrm{N} \quad$ is the cone thermal-conductivity parameter $\left(\mathrm{N}=2 \sigma \mathrm{~T}_{0}^{3} \mathrm{~L} / \lambda\right)$.

## LITERATURE CITED

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2. G. L. Grodzovskii and Z. V. Pasechnik, Prikl. Mat. Tekh. Fiz., No. 3 (1967).
